Stress Concentration of Rectangular Plate with a Hole Made With Composite Material Using Finite Element Analysis

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Abstract: A rectangular plate with a central hole have initiate extensive practical applications in different fields of engineering such as aerospace, marine, automobile and mechanical and is well-known to most engineers. High stress due to discontinuity or abrupt change in geometry is known as stress concentration and is mostly set up at the edges of discontinuity. A stress concentration is typically introduced in plates in the form of circular holes. For the design of plate with a hole, proper knowledge of stresses and stress concentration factor (SCF) at the edge of hole under in plane loading are essential. In this project an attempt is made to review the investigations that have been made on the "stress analysis of a rectangular plate with circular hole". Finite Element simulations using Ansys have been done for stress analysis around the circular hole, made up of different materials. The materials considered are composite material i.e carbon / epoxy and also with mild steel. **Keywords:** Stress concentration factor, Finite element analysis, Carbon / epoxy

I. Introduction

A composite material can be defined as a combination of two or more materials that results in better properties than those of the individual components used alone. In contrast to metallic alloys, each material retains its separate chemical, physical, and mechanical properties. The two constituents are reinforcement and a matrix. The main advantages of composite materials are their high strength and stiffness, combined with low density, when compared with bulk materials, allowing for a weight reduction in the finished part. A fiber has a length that is much greater than its diameter. The length-to-diameter (l/d) ratio is known as the *aspect ratio* and can vary greatly. Continuous fibers have long aspect ratios, while discontinuous fibers have short aspect ratios. Continuous-fiber composites normally have a preferred orientation, while discontinuous fibers generally have a random orientation [5].

The origin of this study lies within the framework of an industrial agreement to study the behaviour of composite materials (carbon/epoxy) used in structural work in the field of aeronautics. In order to anticipate possible problems of mechanical resistance and life span on these materials, a series of static fatigue tests and finite element study have been planned. The objective here is to calculate stress concentration around a circular hole in the composite plate under longitudinal tensile load. The increasing use of composite materials in the design of structural parts with high mechanical performance requires a better understanding and modeling the behavior of these structures. Holes in composites will create stress or strain concentrations and hence will reduce the mechanical properties [1]. A ESPI technique has been used to study the strain concentration round a measurement of the three-dimensional displacement field on the specimen surface. The deformation of the surface of the specimen due to the applied load is seen by a change of the attached random grating [2, 3]. The advantages of the ESPI technique are the accuracy and spatial resolution of the measurement (in the micron meter range), a larger measurement area than the measurement area of strain gages, and a noncontact measurement.

II. Material Selection

Material selection is a step in the process of designing any physical object. In the context of product design, the main goal of material selection is to minimize cost while meeting product performance goals.[4] Systematic selection of the best material for a given application begins with properties and costs of candidate materials. For example, a thermal blanket must have poor thermal conductivity in order to minimize heat transfer for a given temperature difference. The materials chosen consisted of carbon fiber reinforced composites with epoxy resin matrix are shown in table 1.

Fiber Angle	Exx (GPa)	Eyy (GPa)	υxy	υyz	Gxy (GPa)
0^{0}	82.64	9.8034	0.3	0.0356	3.8545
30^{0}	67.2251	14.4947	0.3	0.0356	7.798
45°	53.962	19.943	0.3	0.0356	10.6146
60^{0}	42.9766	26.3346	0.3	0.0356	12.3027
90 ⁰	33.9035	33.9035	0.3	0.0356	12.8642

Table 1: Material Properties of Carbon/Epoxy for different fiber angles

III. Finite Element Analysis

A. Steps involve in Ansys:

Preprocessor: All inputs like element selection, real constrain, material properties (young's modulus, poisons ratio, shear modulus) and meshing of the design is given here. And FE analysis is carried out in the solution.

Solution: The boundary conditions like displacement and axial load are given here. And the results are obtained in general post processor.

General Post Processor:

In post processors the results obtain are

- ➢ Deformation
- Von mises Stress & Strain
- Principle Stress & Strain

B. FE Analysis of Rectangular Plate with Central Hole for Mild Steel

ANSYS 11 has been implemented to solve the present problem considering an axial force of 5KN the element type selected for performing the analysis is quar 4 node 42 material selected was mild steel. The analysis is carried on rectangular plate with central hole of dimensions 200mm * 100mm * 2mm and the diameter of hole is 30mm. The simulated results are shown in figures 1 & 2. Consolidate results are tabulated in table 2.



Fig 1: Von mises stress of mild steel plate



Fig 2: Principle stress of mild steel plate

Ŧ	Ansys	Theoretical	% of Variation
Deformation (mm)	0.142	-	-
Von Stress (N/mm ²)	628	772	18%
Principal Stress (N/mm ²)	583	-	-
Stress Intensity Factor	1.908	2.347	

 Table 2: Ansys & Theoretical results of mild steel plate with % of variation

C. FE Analysis of Rectangular Plate with Central Hole for Carbon/Epoxy

ANSYS 11 has been implemented to solve the present problem considering an axial force of 5KN and the element selected for performing the analysis is Shell element. The analysis is carried on rectangular plate with central hole of dimensions 200mm * 100mm * 4mm and the diameter of hole is 30mm. The simulated results are shown in figures 3 & 4. Consolidate results are tabulated in table 3.



Fig 3: Von mises stress of carbon/epoxy at 90°



Fig 4: Principle stress of carbon/epoxy at 90⁰

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Fiber	Deformation	Principle Stress	Von Mises Stress	Stress Intensity
Angle	(mm)	(N/mm^2)	(N/mm ²)	Factor
0^{0}	2.71	855	929	1.9115
30^{0}	1.95	756	737	1.8658
45^{0}	1.23	623	702	1.9445
60°	0.88	591	682	1.9625
90 ⁰	0.84	579	670	1.956

IV. Results And Discussions

Rectangular plate with central hole of dimensions 200mm * 100mm * 4mm and the hole diameter is analyzed by using carbon epoxy with different fiber angles like 0^0 , 30^0 , 45^0 , 60^0 , 90^0 and found the better fiber angle to determine the stress concentration. The consolidate results are tabulated in table 4 & 5 and the graphs are shown in fig 5 to fig 7.

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Fiber	Deformation	Principle Stress	Von Mises Stress	Stress Intensity	
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60^{0}	0.88	591	682	1.9625	
90^{0}	0.84	579	670	1.956	

Table 4: Consolidate results of different fiber angles of carbon/epoxy



Fig 5: Graphs shows the deformation of plate made Carbon/epoxy at different fiber angles



Fig 6: Graph shows the von mises Stress of plate made Carbon/epoxy at different fiber angles





From the above results the better results are obtained at fiber angle 90^0 in both the materials Carbon/epoxy. Now these results of fiber angle 90^0 are compared with mild steel plate as tabulated in table 5. If we talk about the stress intensity factor fiber angle 30^0 has less concentration compare to other fiber angles as shown in fig 8. And the same 30^0 fiber SIF is less than the MS plate SIF.

Materials	Carbon/epoxy (90 ⁰)	Carbon/epoxy (30 ⁰)	Mild steel
Deformation (mm)	0.84	1.95	0.142
Von Mises Stress (N/mm ²)	670	755	628
Principle Stress (N/mm ²)	579	737	583
Stress Concentration Factor	1.956	1.8658	1.908

Table 5: Comparison results of carbon/epoxy and mild steel

Stress Concentration Factor



Fig 8: Graphs shows the SCF of carbon/epoxy and MS plates

V. Conclusion

In general, the maximum stress concentration is always occurred on hole boundary in a finite width plate with central hole under in-plane static loading. The SCF is maximum at the tip of the hole (perpendicular to loading).

- From the results it is concluded that 30⁰ fiber is the optimum fiber where SCF is less than the other fiber angles.
- Where comes to other angles 0^0 holds good in stress and 90^0 in deflection.
- The suitable optimum fiber angle for rectangular plate with central hole when working with composite materials is 30⁰.

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