

Simulation of Convergent Divergent Rocket Nozzle using CFD Analysis

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Abstract: A rocket nozzle is a mechanical device which is designed to control the rate of flow, speed, direction and pressure of stream that exhaust through it. There are various types of rocket nozzles which are used depending upon the mission of the rocket. This paper contains analysis over a convergent divergent rocket nozzle which is performed by varying the number of divisions in mesh. Also the various contours of nozzle like Cell Equiangle skew, Cell Reynolds number, Pressure, Velocity, Mach Number, and above are calculated at each type of mesh using CFD analysis software ANSYS Fluent.

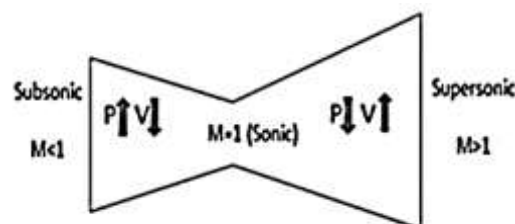
Keywords: ANSYS Fluent, Convergent, Cell Reynolds Number, Mesh, Molecular Prandtl Number.

I. Introduction

A nozzle is a device which is used to give the direction to the gases coming out of the combustion chamber. Nozzle is a tube which has a capacity to convert the thermo-chemical energy generated in the combustion chamber into kinetic energy. The nozzle converts the low velocity, high pressure, high temperature gas in the combustion chamber into high velocity gas of lower pressure and low temperature. A convergent-divergent nozzle is used if the nozzle pressure ratio is high. High performance engines in supersonic aircrafts generally incorporate some form of a convergent-divergent nozzle. Our analysis is carried using softwares like Ansys Workbench for designing of the nozzle and Fluent 15.0 for analyzing the flows in the nozzle. In the present days there is a huge development in Aerospace Engineering for in various prospects. Extensive research is being carried out in the fields like civil and defense prospects. The virtualization is one of the major developments in the field of research, which has revolutionized Aerospace engineering, along with all other branches. The computational techniques are being used widely for getting better results, which are close to experimental techniques. The flow through a convergent-divergent nozzle is one of the benchmark problems used for modelling the compressible flow through computational fluid dynamics. In this paper CFD analysis of a convergent divergent rocket nozzle is done by varying the number of divisions in Mesh and obtaining results for various parameters like pressure, temperature, properties, wall fluxes, Mesh, velocity and adaption.

II. Literature

A convergent-divergent nozzle is designed for attaining speeds that are greater than speed of sound. The design of this nozzle is obtained from the area-velocity relation ($dA/dV = -(A/V)(1-M^2)$) where M is the Mach number (which means the ratio of local speed of flow to the local speed of sound) A is area and V is velocity.



From the above fig we can observe that

a. The decrease in Area results in the increase of pressure and decrease in velocity as seen in the above figure at the entry of the nozzle.

b. The increase in area results in increasing the velocity at the exit of the nozzle by decreasing the pressure.

Also we can find out that

i. $M < 1$ results in subsonic speeds.

ii. $M = 1$ results in sonic speeds.

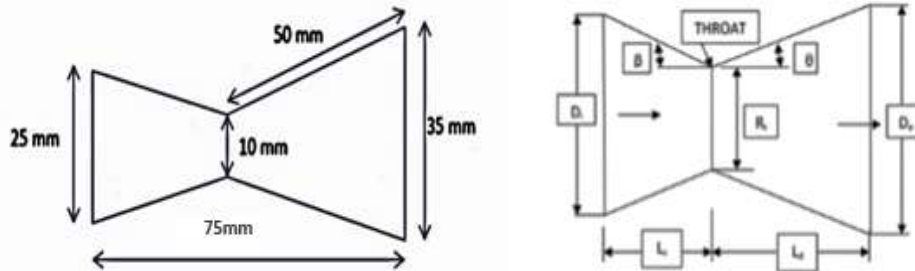
iii. $M > 1$ results in supersonic speeds.

One important point is that to attain supersonic speeds there is a need to maintain favorable pressure ratios across the nozzle.

III. Methodology and implementation

3.1 Modelling

The Geometry of the nozzle was created using ANSYS WORKBENCH 15.0.



Design Parameters

Inlet diameter	25mm
Throat diameter	10mm
Exit diameter	35mm
Pressure	3bar
Ideal gas Viscosity	Sutherland

3.2 Meshing In ANSYS Workbench

After the modelling is completed the meshing is to be done. The module used to perform meshing is Fluid Flow (Fluent). The meshing method used here is Automatic Method and the mesh type is selected as All Quad. The overning equations used in mesh are as follows: It is assumed that there is a unique, single valued relationship between the generalized co-ordinates and the physical co-ordinates which can be expressed as $\eta = \eta(x, y)$ $\xi = \xi(x, y)$

this also implies that,

$$x = x(\xi, \eta) \quad y = y(\xi, \eta)$$

The functional coordinates are determined by the mesh generation process. Given these functional relationships, the governing equations are transformed into corresponding equations containing partial derivatives with respect to the parametric space. For example

$$\frac{\partial T}{\partial x} = \frac{\partial T}{\partial \xi} \frac{\partial \xi}{\partial x} + \frac{\partial T}{\partial \eta} \frac{\partial \eta}{\partial x} \quad (\text{eq.1})$$

$$\frac{\partial T}{\partial y} = \frac{\partial T}{\partial \xi} \frac{\partial \xi}{\partial y} + \frac{\partial T}{\partial \eta} \frac{\partial \eta}{\partial y} \quad (\text{eq.2})$$

The inverse transformation can be written as follows:

$$\frac{\partial T}{\partial \xi} = \frac{\partial T}{\partial x} \frac{\partial x}{\partial \xi} + \frac{\partial T}{\partial y} \frac{\partial y}{\partial \xi} \quad (\text{eq.3})$$

$$\frac{\partial T}{\partial \eta} = \frac{\partial T}{\partial x} \frac{\partial x}{\partial \eta} + \frac{\partial T}{\partial y} \frac{\partial y}{\partial \eta} \quad (\text{eq.4})$$

The Poisson Equation that is solved is of the form as in the following equations :

$$\frac{\partial^2 \xi}{\partial x^2} + \frac{\partial^2 \xi}{\partial y^2} = P(\xi, \eta)$$

$$\frac{\partial^2 \eta}{\partial x^2} + \frac{\partial^2 \eta}{\partial y^2} = Q(\xi, \eta)$$

Where P and Q are predefined functions that are used to control grid clustering. Here in this project Meshing plays a main role, since we are obtaining results by varying the Number of divisions in mesh. The number of divisions are varied at the vertical surfaces (inlet and exit) and the inclined surfaces (walls).

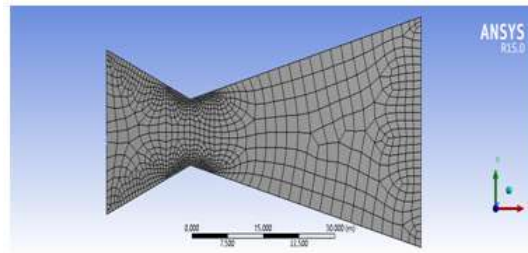
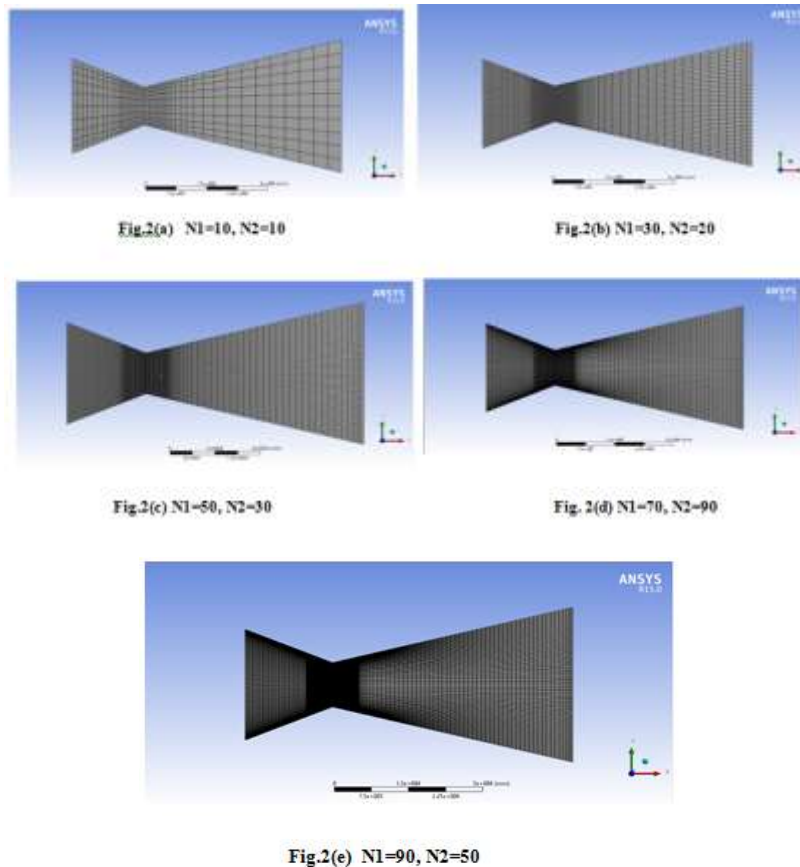


Fig. 1

The mesh obtained initially will be unstructured mesh (fig.1) and this cannot be used to obtain accurate results. Since the edges are prismatic the mesh can be converted into structured meshing by using Mapped Face Meshing. The analysis is done for five types of meshes which are obtained by varying the number of divisions in mesh. The variation of the number of divisions is done on the inlet, exit and on the walls of the nozzle. The following is the nomenclature that is followed to mention the Number of divisions.
 N1-Number of divisions on the Inlet and exit of the nozzle
 N2-Number of divisions on the inclined walls of the nozzle.



The following table provides the information about the type of mesh associated with a particular type of number of mesh divisions over a surface.

Mesh 1	Mesh 2	Mesh 3	Mesh 4	Mesh 5
N1=10	N1=30	N1=50	N1=70	N1=90
N2=10	N2=20	N2=30	N2=90	N2=50

From the above we can observe that the number of divisions at N1 are being varied by 20, whereas at N2 the variation is 10 for each mesh.

3.3 Boundary Conditions

1. Mass flow inlet
2. Outlet
3. Walls

Specification of the boundary zones has to be done in WORKBENCH only, as there is no possibility to specify the boundary zones in FLUENT. Therefore proper care has to be taken while defining the boundary conditions in WORKBENCH. With all the zones defined properly the mesh is exported to the solver. The solver used in this problem is ANSYS FLUENT. The exported mesh file is read in Fluent for solving the problem.

3.4 Solving

FLUENT analysis is carried out on nozzle at different meshing conditions.

Analysis Procedure

The same procedure is followed for all the 5 types of mesh and the results are validated.

PROCEDURE	DETAILS
Problem Setup General-Solver	Type: Density based Velocity : Absolute Time: Steady 2D space: Planar
Models	Energy : On Viscous: Laminar
Materials	Fluid : Air Density: Ideal Gas Viscosity: Sutherland
Boundary conditions	Inlet : Pressure Inlet Gauge Total Pressure (pa): 3e5 Outlet : Pressure Out let Gauge Pressure(pa): 0
Reference Values	Compute from : Inlet Reference Zone : Solid Surface body
Monitors	Create-walls -cd 1 Select Print to console and plot.
Initialization	Standard Initialization Compute from Inlet.
Solution	Solution Controls – Courant Number=5. Run Calculation: Enter the Number of iterations, Click calculation.

IV. Results And Discussion

4.1 Convergence History

The following graphs present the convergence history of the nozzle at different types of meshes.

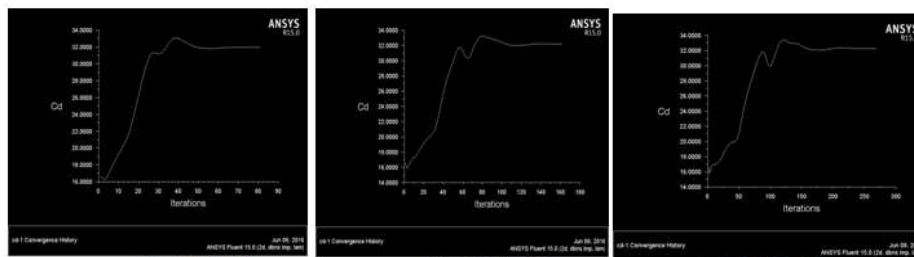


Fig.3(a) Mesh 1

Fig.3(b) Mesh 2

Fig. 3(c) Mesh 3

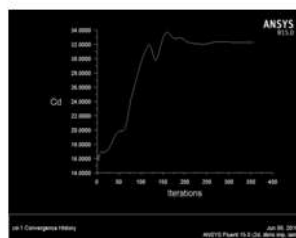


Fig. 3(d) Mesh 4

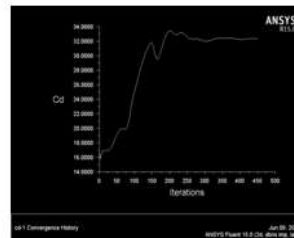
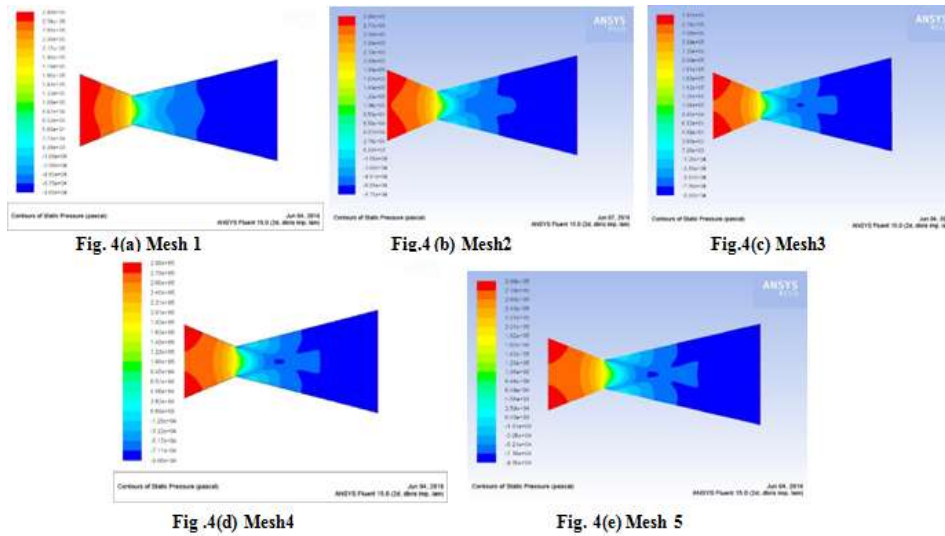


Fig.3 (e) Mesh 5

The solution is converged after 83 iterations for mesh1, 165 iterations for mesh2, 268 iterations for mesh 3, 342 iterations for mesh 4, and 440 iterations for mesh 5.

4.2 Static Pressure

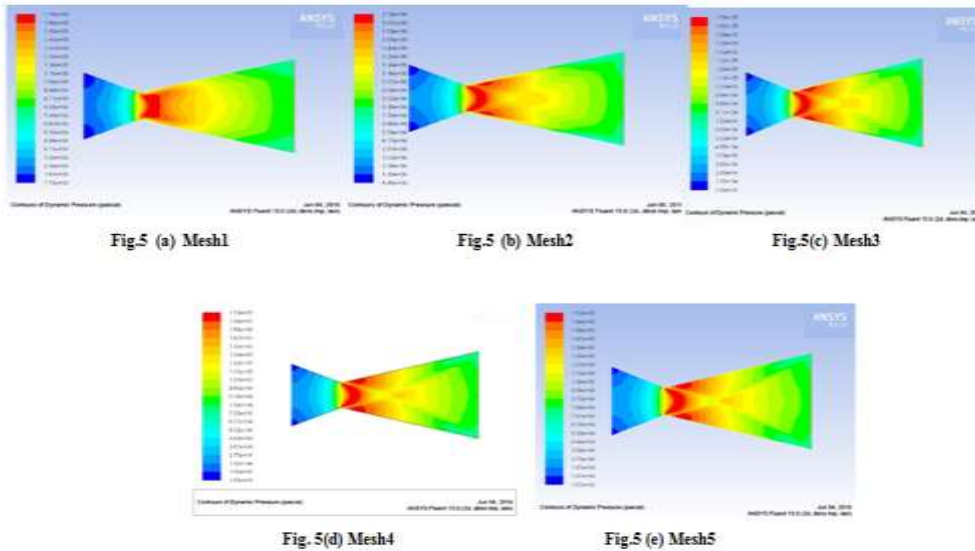


The results obtained for the static pressure at each type of mesh are as follows.

Mesh	Min value (Pa)	Max value (Pa)
1	-86476.05	292786.7
2	-87471.49	295932.2
3	-89273.85	296949.1
4	-90591.82	298549.4
5	-91049.52	298767.8

The maximum value of the static pressure increases from Mesh 1 to Mesh 5 gradually. The maximum value at mesh1 is 292786.7 and the value is increased at mesh 5 as 298767.8

4.3 Dynamic Pressure

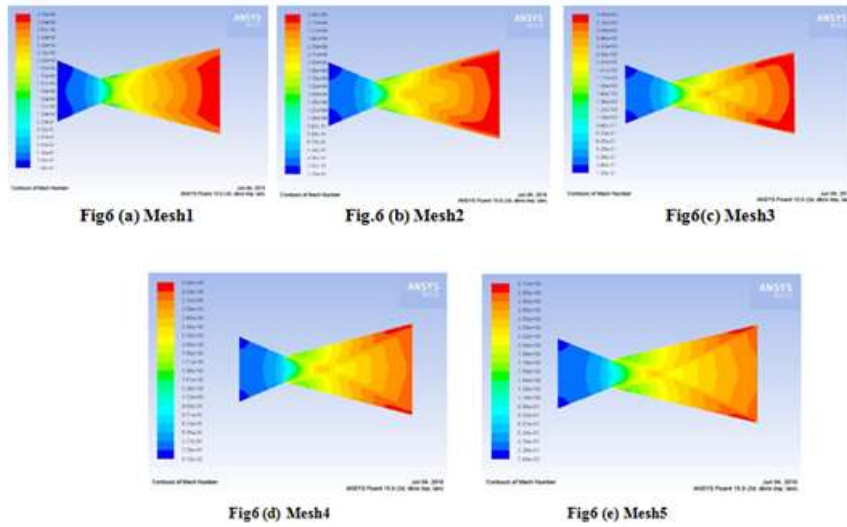


The results obtained for the dynamic pressure over each type of mesh are as follows:

Mesh	Min value (Pa)	Max Value (Pa)
1	7786.104	174392.5
2	4451.948	175928.7
3	3340.72	176164.5
4	1845.276	172981.3
5	1572.161	172838.4

As we observe the minimum value of the dynamic pressure decreases gradually on the increase in the number of mesh divisions.

4.4 Mach Number



The obtained results for the Mach number on the variation of mesh size are as follows:

Mesh	Min	Max
1	0.1677308	2.784976
2	0.1264253	2.844711
3	0.1094085	2.878834
4	0.08119585	3.038563
5	0.074926	3.105635

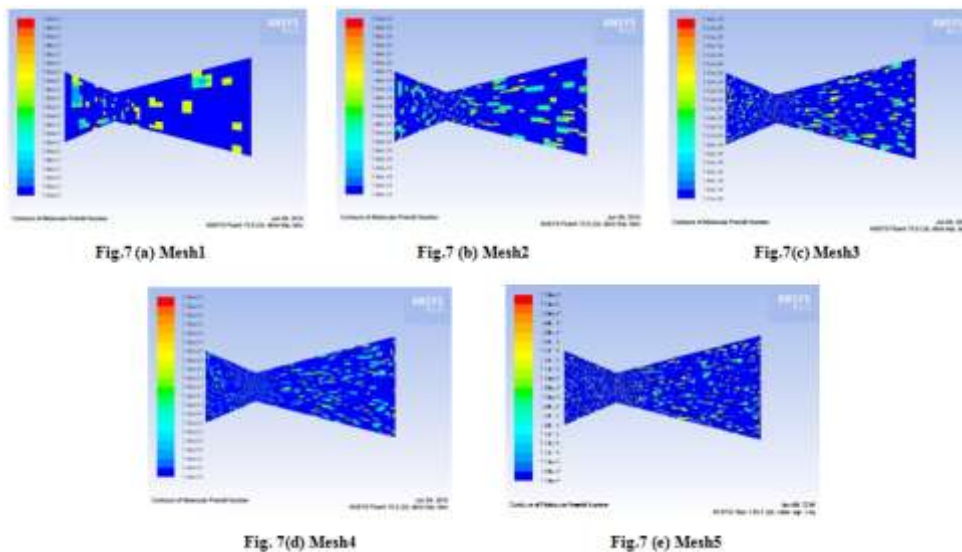
The Mach number on increase in the number of mesh divisions shows a considerable increase in its value from 2.784976 to 3.105635 as shown in the above table.

4.5 Molecular Prandtl Number

The following are the results obtained for the molecular prandtl number:

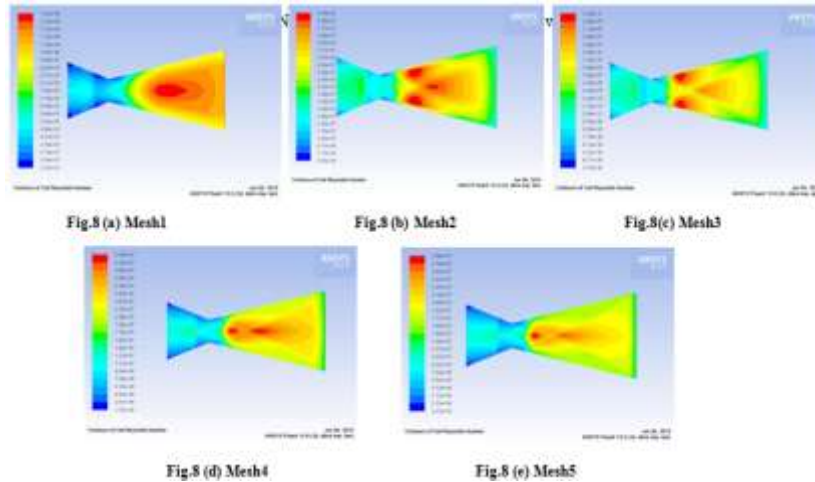
Mesh	Min	Max
1	0.7441759	0.7441761
2	0.7441759	0.7441762
3	0.7441759	0.7441762
4	0.7441759	0.7441762
5	0.7441759	0.7441762

The contours of the Molecular Prandtl Number are shown below.



4.6 Cell Reynolds Number

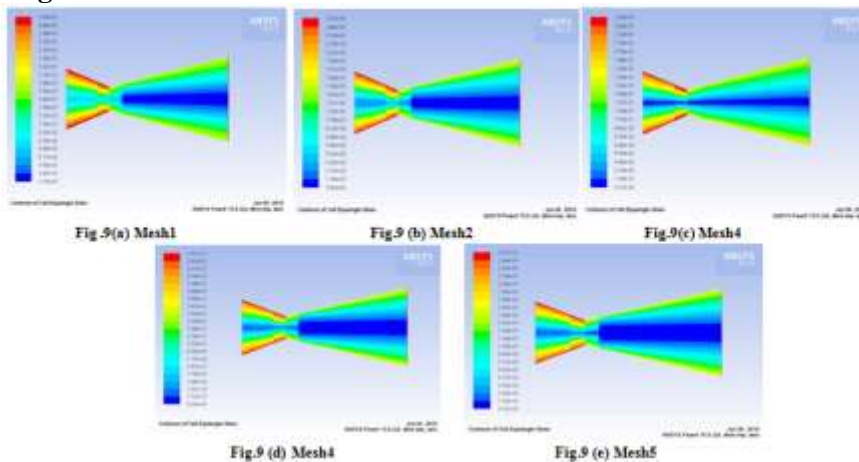
the contours of Cell Reynolds Number are presented above. The following table elucidates the results obtained at



The contours of Cell Reynolds Number are presented above. The results obtained are elucidated below in the following table.

Mesh	Min	Max
1	2.072836e+07	1.323412e+08
2	6348036	3.400297e+07
3	3764984	2.389734e+07
4	1346941	3.38988e+07
5	922478.5	2.892826e+07

4.7 Cell Equi angle Skew



The following are the results obtained for the cell equi angle skew after solving in Fluent.

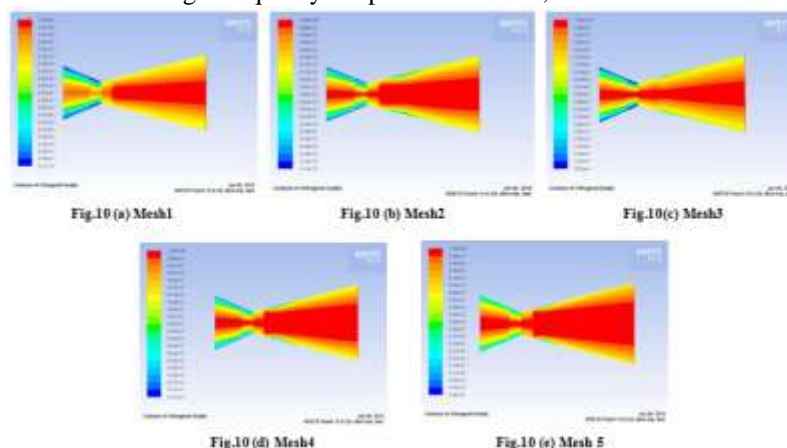
Mesh	Min	Max
1	0.02246387	0.2575587
2	0.004999726	0.2570593
3	0.006369548	0.2571762
4	0.0005017999	0.2571143
5	5.518553e-05	0.2571229

4.8 Orthogonal Quality

The results obtained for the orthogonal quality are as follows,

Mesh	Min	Max
1	0.9205856	0.9993722
2	0.9178777	0.999963
3	0.9197189	0.9999467
4	0.9126614	0.9999996
5	0.8950424	1

As we observe the max orthogonal quality remains almost the same on the variation of the mesh divisions. The contours of the orthogonal quality are presented below,



V. Conclusion

A nozzle model was developed to determine the pressure, Mach number, cell equi angle skew, cell Reynolds number, molecular prandtl number and orthogonal quality in it. Various steps of the model were validated with good accordance with the experimental data and numerical results found in the literature. The contours of the above mentioned parameters are found after analyzing the model successfully in the solver. Also the minimum and maximum values of all the parameters at all the five types of mesh are tabulated.

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