

# Study Of Quasi -Static Thermoelastic Steady State Behavior Of Thick Circular Plate

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## Abstract

The present paper deals with the determination of displacement and thermal stresses in a thick circular plate with internal heat generation. Arbitrary heat flux is applied at the upper surface of a thick circular plate, whereas lower surface at zero temperature and the fixed circular edge is thermally insulated. Here we compute the effects of internal heat generation of a thick circular plate in terms of stresses along radial direction. The governing heat conduction equation has been solved by the method of integral transform technique. The results are obtained in a series form in terms of Bessel's functions. The results for temperature change and stresses have been computed numerically and illustrated graphically.

**Keywords:** Internal heat generation, steady state, thermal stresses.

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## I. Introduction

The steady state thermal stresses in circular disk subjected to an axisymmetric temperature distribution on the upper face with zero temperature on the lower face and the circular edge has been considered by Nowacki(1957). Sharma et al. (2004) studied the behavior of thermoelastic thick plate under lateral loads. Kulkarni and Deshmukh (2008) determined quasi-static thermal stresses in steady state thick circular plate. Deshmukh *et al.* (2009) studied non homogeneous steady state heat conduction problem in a thin circular plate and discussed its thermal stresses due to its internal heat generation at a constant rate. Bhongade and Durge (2013) considered thick circular plate and discuss, effect of Michell function on steady state behavior of thick circular plate. In this paper thick circular plate is considered and discussed its thermoelasticity with the help of the Goodier's thermoelastic displacement potential function and the Michell's function. To obtain the temperature distribution integral transform method is applied. The results are obtained in series form in terms of Bessel's functions and the temperature change and stresses have been computed numerically and illustrated graphically. Here we compute the effects of internal heat generation in terms of stresses along radial direction. A mathematical model has been constructed of a thick circular plate with the help of numerical illustration by considering aluminum (pure) circular plate. No one previously studied such type of problem. This is new contribution to the field.

The direct problem is very important in view of its relevance to various industrial mechanics subjected to heating such as the main shaft of lathe, turbines and the role of rolling mill, base of furnace of boiler of a thermal power plant, gas power plant.

## Formulation of the problem

Consider a thick circular plate of thickness  $h$  defined by  $0 \leq z \leq h$ . The initial temperature in a thick circular plate is zero. The arbitrary heat flux  $q_0$  is applied over the upper surface ( $z = h$ ) and the lower surface ( $z = 0$ ) is at temperature zero. The fixed circular edge is thermally insulated. Assume the circular boundary of a thick circular plate is free from traction. Under these prescribed conditions, the thermal steady state temperature, displacement and stresses in a thick circular plate with internal heat generation are required to be determined. The differential equation governing the displacement potential function is given in (2003) as

(1)

where  $K$  is the restraint coefficient and temperature change is initial(ambient) temperature. Displacement function is known as Goodier's thermoelastic displacement potential.

The steady state temperature of the plate satisfies the heat conduction equation,

(2)

with the boundary conditions

(3)

(4)

(5)

where  $k$  is the thermal conductivity of the material of the plate and  $q$  is the internal heat generation.  
The Michell's function  $M$  must satisfy

(6)

where

(7)

The components of the stresses are represented by the thermoelastic displacement potential and Michell's function  $M$  as

(8)

(9)

(10)

and

(11)

where  $G$  and  $\nu$  are the shear modulus and Poisson's ratio respectively.

For traction free surface stress functions

(12)

### **SOLUTION**

To obtain the expression for temperature  $T(r, z)$ , we introduce the finite Hankel transform over the variable  $r$  and its inverse transform defined by (1968) as

(13)

(14)

where

(15)

Eigen value are the positive root of

(16)

are roots of the transcendental equation

where  $J_n$  is Bessel function of the first kind of order  $n$ .

On applying the finite Hankel transform defined in the Eq. (13) and its inverse transform defined in Eq. (14) to the Eq. (2), one obtains the expression for temperature as

{ (17)

is particular integral of differential Eq. (2).

Michells function  $M$

Now suitable form of  $M$  which satisfy Eq. (6) is given by

(18)

where are arbitrary functions.

Goodiers thermoelastic displacement potential

Assuming the displacement function which satisfies Eq. (1) as

(19)

Now using Eqs. (17), (18) and (19) in Eqs. (8), (9), (10) and (11), one obtains the expressions for stresses respectively as

{

(20)

{

}

(21)

} (22)

(23)

In order to satisfy condition Eq. (12), solving Eqs. (20), (22) and (23) for and one obtains

Let

(24)

(25)

## II. Special Case and Numerical Calculations

Setting

(1)

where is well known direct delta function of argument  $r$ .

(2)

=

Material properties

The numerical calculation has been carried out for aluminum (pure) circular plate with the material properties defined as

Thermal diffusivity = 84.18

Specific heat

Thermal conductivity  $k = 204.2$  W/m K,

Shear modulus

Poisson ratio

Roots of transcendental equation

The are the roots of transcendental equation The numerical calculation and the graph has been carried out with the help of mathematical software Mat lab.

## III. Discussion

In this paper a thick circular plate is considered and determined the expressions for temperature and stresses due to internal heat generation within it and we compute the effects of internal heat generation in terms of stresses along radial direction by substituting in Eqs. (17), (19), (20), (21), (22), (23), (24) and (25) and we compare the results for and . As a special case mathematical model is constructed for considering aluminum (pure) circular plate with the material properties specified above.

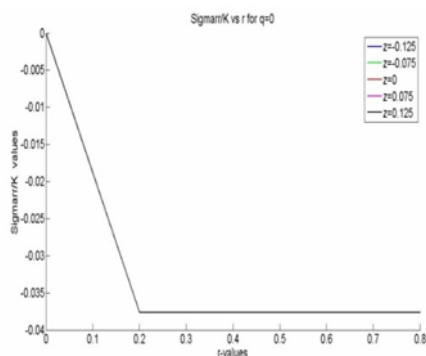


Figure 1 Radial stress function for  $(q = 0)$ .

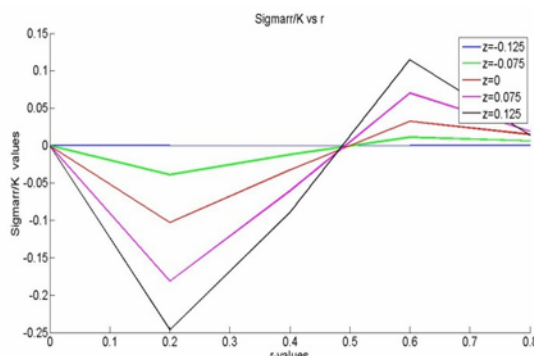


Figure 2 Radial stress function for  $(q_0)$ .

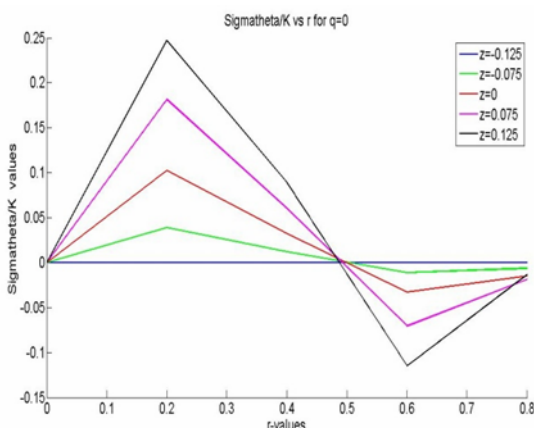


Figure 3 Angular stress function for  $(q = 0)$ .

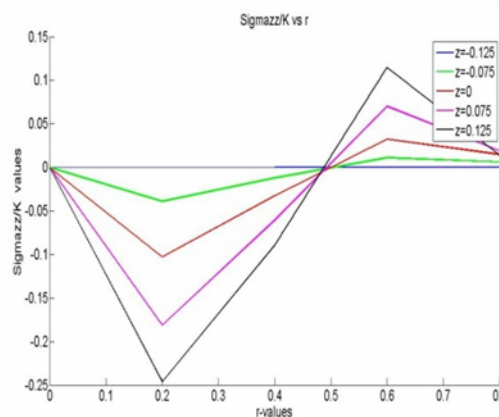


Figure 4 Angular stress function for  $(q_0)$ .

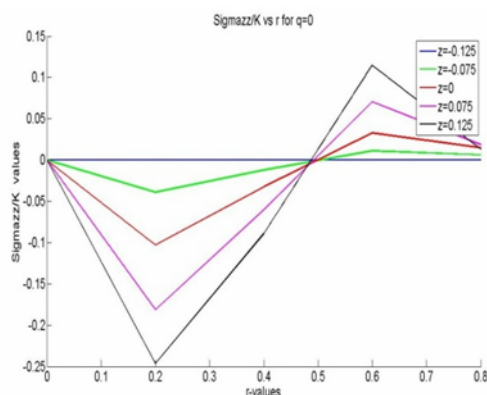


Figure 5 Axial stress function for ( $q = 0$ ).

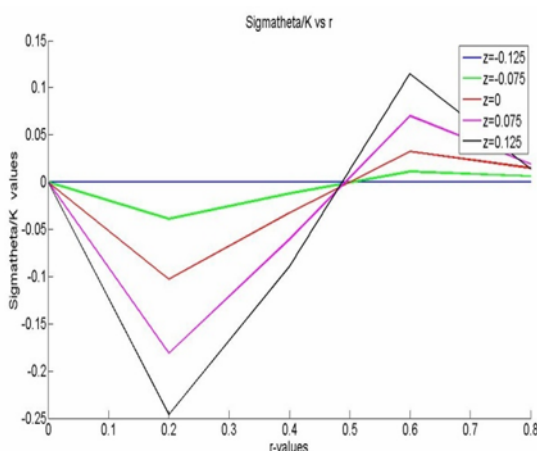


Figure 6 Axial stress function for ( $q = 0$ ).

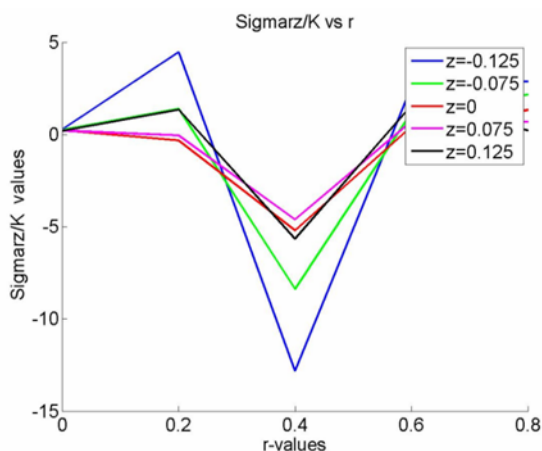


Figure 7 Stress function for ( $q = 0$ ).

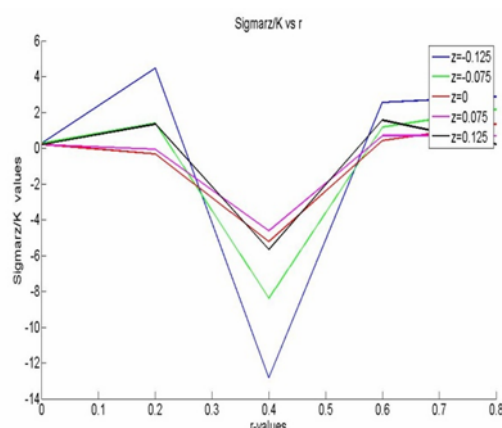


Figure 8 Stress function for ( $q = 0$ ).

From figure 1 and 2, it is observed that the radial stress function develops compressive stress for ( $q = 0$ ) in the radial direction whereas due to internal heat generation the radial stress function develops tensile stress in the radial direction.

From figure 3 and 4, it is observed that the angular stress function develops compressive stress for ( $q = 0$ ) in the radial direction whereas due to internal heat generation the angular stress function develops tensile stress in the radial direction.

From figure 5 and 6, it is observed that the axial stress function develops tensile stress for ( $q = 0$ ) and ( $q = 0$ ) in the radial direction.

From figure 7 and 8, it is observed that the stress function develops tensile stress for ( $q = 0$ ) and ( $q = 0$ ) in the radial direction.

#### IV. Conclusion

We can conclude that due to internal heat generation in thick circular plate the radial stresses and the angular stresses are tensile. Also, it can be observed that there is no effect of internal heat generation on axial stress function and stress function in the radial direction.

The results obtained here are useful in engineering problems particularly in the determination of state of stress in a thick circular plate and base of furnace of boiler of a thermal power plant and gas power plant.

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